| Name: <br> Enrolment No: |  |  |  |
| :---: | :---: | :---: | :---: |
| \left.UPES  <br> End Semester Examination, December 2023 $\right)$ |  |  |  |
| $\begin{gathered} \text { SECTION A } \\ \text { (5Qx4M=20Marks) } \end{gathered}$ |  |  |  |
| S. <br> No. |  | Ma rks | CO |
| Q 1 | Read these statements and answer the question that follow <br> Raghav: All inviscid flows are irrotational. <br> Raghunath: All irrotational flows are inviscid. <br> Raghuram: All viscous flows are rotational. <br> Raman: All viscous flows are irrotational. <br> Ramesh: All irrotational flows are inviscid. <br> Who is/are correct? Justify your answer with examples. (None/All of the choices may also be correct) | 4 | CO1 |
| Q 2 | Automobiles need to breathe in air for purposes such as cooling of engines. This may lead to an additional 'cooling drag' on the automobiles The pictures and the plots below represent this phenomenon: <br> The figure above shows two set up of experiments conducted for which drag coefficient was estimated. The difference between the drag coefficient is shown below: | 4 | CO1 |



| Q 7 | For flow over a hypothetical flat plate of length $L$, the velocity profile can be approximated as $\frac{u}{U}=0.7 \frac{y}{\delta}$ <br> Find: <br> a) Boundary layer thickness at a distance $x$. <br> b) Shear stress at a distance x <br> c) Local drag coefficient <br> d) Coefficient of Drag | 10 | CO2 |
| :---: | :---: | :---: | :---: |
| Q 8 | Gasoline flows in a long, underground pipeline at a constant temperature of 15 degree Celsius. Two pumping stations at the same elevation are located 13 km apart. The pressure drop between the stations is 1.4 MPa . The pipeline is made from $0.6-\mathrm{m}-$ diameter pipe. Although the pipe is made from commercial steel, age and corrosion have raised the pipe roughness to approximately that for galvanized iron. Compute the volume flow rate. How long will it take for the Gasoline flowing through such a pipeline to fill a $10,000 \mathrm{~L}$ tank? <br> OR <br> A high school project involves building a model ultralight airplane. Some of the students propose making an air foil from a sheet of plastic 5 m long 3.7 m wide at an angle of attack of 10 degrees. At this air foil's aspect ratio and angle of attack the lift and drag coefficients are $C_{L}=0.75$ and $C_{D}=0.19$. If the airplane is designed to fly at $40 \mathrm{~m} / \mathrm{s}$, what is the maximum total payload? What will be the required power to maintain flight? Does this proposal seem feasible? | 10 | CO 3 |
| Q 9 | Two immiscible fluids of equal density are flowing down a surface inclined at a 60 degree angle. The two fluid layers are of equal thickness $\mathrm{h}=10 \mathrm{~mm}$; the kinematic viscosity of the upper fluid is $1 / 5^{\text {th }}$ that of the lower fluid, which is V lower $=0.01$ $\mathrm{m}^{2} / \mathrm{s}$. Find the velocity at the interface and the velocity at the free surface. Plot the velocity distribution. | 10 | CO4 |
| $\begin{gathered} \text { SECTION-C } \\ \text { (2Qx20M=40 Marks) } \end{gathered}$ |  |  |  |
| Q 10 | Using Continuity and Navier-Stokes Equation in cylindrical coordinates for fluid flow between two parallel plates, derive expressions for <br> a) Velocity profile between two plates spaced distance $D$ apart. <br> b) Relationship between discharge and pressure drop over length $L$ of the plates. | 20 | CO 3 |
| Q 11 | The entrance region of a parallel, rectangular duct flow is shown in figure. The duct has a width W and height H , where $\mathrm{W} \gg \mathrm{H}$. The fluid density $\boldsymbol{\rho}$ is constant, and the flow is steady. The velocity variation in the boundary layer of thickness $\delta$ at station is assumed to be linear, and the pressure at any cross- section is uniform. <br> (a) Using the continuity equation, shows that $U_{1} / U_{2}=1-\delta / \mathrm{H}$. <br> (b) Find the pressure coefficient $C_{p}=\left(p_{1}-p_{2}\right) /\left(\frac{1}{2} \rho U_{1}{ }^{2}\right)$ <br> (c) Show that | 20 | CO4 |



## Appendix

## Haaland Equation :

$\frac{1}{\sqrt{f}}=-1.8 \log \left[\frac{6.9}{\boldsymbol{R e}}+\left(\frac{\varepsilon / D}{3.7}\right)^{1.11}\right]$

## Potential Flows:

| Description of Flow Field | Velocity Potential | Stream Function | Velocity <br> Components ${ }^{\text {a }}$ |
| :---: | :---: | :---: | :---: |
| Uniform flow at angle $\alpha$ with the $x$ axis (see Fig. 6.16b) | $\phi=U(x \cos \alpha+y \sin \alpha)$ | $\psi=U(y \cos \alpha-x \sin \alpha)$ | $\begin{aligned} & u=U \cos \alpha \\ & v=U \sin \alpha \end{aligned}$ |
| $\begin{aligned} & \text { Source or sink } \\ & \text { (see Fig. 6.17) } \\ & m>0 \text { source } \\ & m<0 \text { sink } \end{aligned}$ | $\phi=\frac{m}{2 \pi} \ln r$ | $\psi=\frac{m}{2 \pi} \theta$ | $\begin{aligned} & v_{r}=\frac{m}{2 \pi r} \\ & v_{\theta}=0 \end{aligned}$ |
| ```Free vortex (see Fig. 6.18) \(\Gamma>0\) counterclockwise motion \(\Gamma<0\) clockwise motion``` | $\phi=\frac{\Gamma}{2 \pi} \theta$ | $\psi=-\frac{\Gamma}{2 \pi} \ln r$ | $\begin{aligned} & v_{r}=0 \\ & v_{\theta}=\frac{\Gamma}{2 \pi r} \end{aligned}$ |
| Doublet (see Fig. 6.23) | $\phi=\frac{K \cos \theta}{r}$ | $\psi=-\frac{K \sin \theta}{r}$ | $\begin{aligned} & v_{r}=-\frac{K \cos \theta}{r^{2}} \\ & v_{\theta}=-\frac{K \sin \theta}{r^{2}} \end{aligned}$ |

Momentum Equations in Cartesian Coordinates:

$$
\begin{aligned}
& \boldsymbol{\rho}\left(\frac{\partial U}{\partial t}+U \frac{\partial U}{\partial x}+V \frac{\partial U}{\partial y}+W \frac{\partial U}{\partial z}\right)=-\frac{\partial P}{\partial x}+\boldsymbol{\rho} g_{x}+\mu\left(\frac{\partial^{2} U}{\partial x^{2}}+\frac{\partial^{2} U}{\partial y^{2}}+\frac{\partial^{2} U}{\partial z^{2}}\right) \\
& \boldsymbol{\rho}\left(\frac{\partial V}{\partial t}+U \frac{\partial V}{\partial x}+V \frac{\partial V}{\partial y}+W \frac{\partial V}{\partial z}\right)=-\frac{\partial P}{\partial y}+\boldsymbol{\rho} g_{y}+\mu\left(\frac{\partial^{2} V}{\partial x^{2}}+\frac{\partial^{2} V}{\partial y^{2}}+\frac{\partial^{2} V}{\partial z^{2}}\right) \\
& \boldsymbol{\rho}\left(\frac{\partial W}{\partial t}+U \frac{\partial W}{\partial x}+V \frac{\partial W}{\partial y}+W \frac{\partial W}{\partial z}\right)=-\frac{\partial P}{\partial z}+\boldsymbol{\rho} g_{z}+\mu\left(\frac{\partial^{2} W}{\partial x^{2}}+\frac{\partial^{2} W}{\partial y^{2}}+\frac{\partial^{2} W}{\partial z^{2}}\right)
\end{aligned}
$$

