| Name: <br> Enrolment No: |  |  |  |
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| UPESSemester Examination, December 2023 |  |  |  |
| Course: Vehicle Dynamics <br> Program: M.Tech E-Mobility <br> Course Code: MEEM7003 |  | Semester: I <br> Time: 03 hrs <br> Max. Marks: 100 |  |
| Instructions: Wherever applicable, must draw appropriate free body diagram and work with symbols before substituting numerical values. |  |  |  |
| $\begin{gathered} \text { SECTION A } \\ (5 Q \times 4 M=20 M a r k s) \end{gathered}$ |  |  |  |
| S. No. |  | Marks | CO |
| Q 1 | Explain the response of an underdamped system for free vibrations and logarithmic decrement. | 4 | CO1 |
| Q 2 | Explain slip angle. Discuss why it is required during turning? | 4 | CO1 |
| Q 3 | Explain various sources that cause vibrations in a vehicle. | 4 | CO1 |
| Q 4 | Explain the 'angle of rolling down' for a passenger car parked on a banked road along with formula. (Note: road is not inclined but is banked) | 4 | CO1 |
| Q 5 | Analyze the behavior of an oversteer and understeer vehicle by plotting 'steer angle vs speed' curve for both. Analyze how the steer angle should be changed for each while negotiating a constant radius curve if the driver also accelerates during the turn. | 4 | CO 2 |
| $\begin{gathered} \text { SECTION B } \\ \text { (4Qx10M=40 Marks) } \end{gathered}$ |  |  |  |
| Q 6 | A car with mass $=1500 \mathrm{~kg}$, wheel base $=3 \mathrm{~m}$, has $60 \%$ of weight distribution on front tires. Lateral stiffness of front and rear tires is $C_{f}=$ $40 \mathrm{kN} / \mathrm{rad}$ and $C_{r}=45 \mathrm{kN} / \mathrm{rad}$. Calculate the understeer coefficient and critical speed or characteristic speed as applicable. | 10 | $\mathrm{CO3}$ |
| Q 7 | Analyze the reason for rolling resistance. Also derive an expression for effective radius (also called rolling radius) of tire. | 10 | CO2 |
| Q 8 | A car has $55 \%$ of static load on the rear axle. The ratio of height of CG to wheel base, $\mathrm{h} / \mathrm{L}=0.2$. Coefficient of friction $\mu=0.8$, coefficient of rolling resistance $f_{r}=0.01$. Determine the ideal brake force distribution for which the system should be designed. (That is $K_{b f}$ and $K_{b r}$ ) | 10 | $\mathrm{CO3}$ |
| Q 9 | Design a suitable vibration absorber for a machine that has mass of 50 kg and runs at 6000 rpm . Its forcing frequency is very near to its natural | 10 | $\mathrm{CO3}$ |


|  | frequency. The nearest natural frequency of the 2 DOF system is to be at least $20 \%$ from the forced frequency. <br> OR <br> Determine the parameters in an equivalent system model of the system when $\theta$, the clockwise angular displacement of the bar from the system's equilibrium position, is used as the generalized coordinate. |  |  |
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|  | $\begin{gathered} \text { SECTION-C } \\ (2 \mathrm{Q} \times 20 \mathrm{M}=40 \text { Marks }) \end{gathered}$ |  |  |
| Q 10 | Gopal, as a dedicated cop, is doing his best to chase a suspect. He is driving a 'rear wheel drive' car uphill and using the maximum available traction. He is only able to accelerate at $1 \mathrm{~m} / \mathrm{s}^{2}$. The relevant data is the following - <br> Mass of car, $\mathrm{m}=1000 \mathrm{~kg}$ <br> Slope angle, $\theta=15^{\circ}$ <br> Speed, v=10 m/s <br> Car frontal area, $A_{f}=2 \mathrm{~m}^{2}$ <br> Coefficient of drag, $C_{D}=0.3$ <br> Coefficient of rolling resistance, $f_{r}=0.01$ <br> Wheel base, $\mathrm{L}=3 \mathrm{~m}$ <br> Distance of CG from front axle, $l_{1}=1.5 \mathrm{~m}$ <br> Height of CG, $\mathrm{h}=0.3 \mathrm{~m}$ <br> Use $\mathrm{g}=10 \mathrm{~m} / \mathrm{s}^{2}$ <br> Assume aerodynamic drag is acting on CG and density of air is $1.2 \mathrm{~kg} / \mathrm{m}^{3}$. <br> Determine/do <br> (a) Draw a FBD showing all forces <br> (b) The load on front and rear axle | 20 | CO 4 |


|  | (c) The available traction force <br> (d) Evaluate the coefficient of friction at road-tire interface. Analyze if the value is reasonable. |  |  |
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| Q 11 | A vehicle is modeled using a quarter car model. The relevant data is the following - <br> Sprung mass, $m_{s}=2000 \mathrm{~kg}$ <br> Unsprung mass, $m_{u s}=200 \mathrm{~kg}$ <br> Suspension system stiffness, $k_{s}=100 \mathrm{kN} / \mathrm{m}$ <br> Tire stiffness, $k_{t}=1000 \mathrm{kN} / \mathrm{m}$ <br> Damping coefficients $c_{s}, c_{t}$ are negligible. (Use them in the formulation and eventually make them zero). <br> Determine the natural frequencies using (a) matrix method and (b) approximate method and compare them. <br> OR <br> A vehicle is modeled using the pitch \& bounce model. The relevant data is the following - <br> Sprung mass, $m_{s}=2000 \mathrm{~kg}$ <br> Radius of gyration, $r_{y}=1.25 \mathrm{~m}$ <br> Distance from front axle to $\mathrm{CG}=1.25 \mathrm{~m}$ <br> Distance from rear axle to $\mathrm{CG}=1.5 \mathrm{~m}$ <br> Front spring stiffness, $k_{f}=35 \mathrm{kN} / \mathrm{m}$ <br> Rear spring stiffness, $k_{r}=40 \mathrm{kN} / \mathrm{m}$ <br> Determine/Do <br> (a) Derive the equations of motion for the 2 DOF system <br> (b) Determine the natural frequencies and mode shapes <br> (c) Calculate the locations of oscillation centers | 20 | $\mathrm{CO3}$ |

