Name:

**Enrolment No:** 

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## UNIVERSITY OF PETROLEUM AND ENERGY STUDIES

**End Semester Examination, December 2018** 

Programme Name: B.Tech MechanicalCourse Name: Solar Thermal TechnologiesCourse Code: MHEG 451Nos. of page(s): 4

Semester : VII Time : 03 hrs Max. Marks : 100

Instructions: Assume the suitable data if required SECTION A

S. No.		Marks	CO
Q 1	Differentiate between the Flat plate solar collectors and concentrating solar collectors.	4	CO2
Q 2	Explain thermo-syphon system for water heating.	4	CO3
Q 3	Describe the solar thermal air heating system for domestic house.	4	CO1
Q 4	Why the orientation is needed in concentrating type collectors?	4	CO1
Q 5	A compound parabolic collector, 1 m long, has an acceptance angle of $20^{\circ}$ . The absorber surface of the collector is flat and has a width of 10 cm. Calculate the concentration ratio, the aperture, the height and the surface area of the concentrator.	4	CO4
	SECTION B	I I	
Q 6	Describe the solar thermal power plant with a neat diagram.	10	CO1
Q 7	Explain the working of solar refrigeration cycle with a neat diagram.	10	C05
Q 8	Explain the working principle of compound parabolic collector with a neat sketch.	10	CO2
Q 9	Classify the concentrated solar collectors and mention its applications		
	(OR) Explain the working of solar cookers.	10	CO2 CO3

	SECTION	-C		
Q 10	Explain the working of parabolic trough collector			
	Discuss the limitations of its concentration ratio and	nd derive an equation for useful	20	CO3
	heat gain.			
Q 11	Calculate the overall heat loss coefficient $U_1$ for parabolic concentrating collector system. The record coated absorber tube with one glass cover around it.	eiver consists of a selectively –		
	Absorber tube inner diameter	: 7.5 cm		
	Absorber tube outer diameter	: 8.1 cm		
	Glass cover inner diameter	: 14.4 cm		
	Glass cover outer diameter	: 15.0 cm		
	Emissivity of absorber tube surface	: 0.15		
	Emissivity of glass	: 0.88		
	Mean temperature of absorber tube	: 170°C		
	Ambient temperature	: 25°C		
	Wind speed	: 4 m/s		
	(OR) A CPC is mounted on a horizontal E - W axis and sloping at an angle of 40°. The concentration ratio of its absorber tube plate is 6 cm and its length is 2 m.	f the collector is 6.5, the width of		CO
	A CPC is mounted on a horizontal E - W axis and sloping at an angle of $40^{\circ}$ . The concentration ratio of its absorber tube plate is 6 cm and its length is 2 m. a fluid (Cp = 2.35 kJ/kg-K) which enters at a tem exit temperature of the fluid and the instantaneous statement of the fluid and the instantaneous statement.	f the collector is 6.5, the width of The collector is used for heating perature of 130°C. Calculate the	20	CO
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Equations for cylindrical parabolic concentrating collector.

1) Useful heat for unit length

$$\frac{q_l}{L} = h_{p-c}(T_{pm} - T_c)\pi D_o + \frac{\sigma \pi D_o(T_{pm}^4 - T_c^4)}{\left\{\frac{1}{\varepsilon_p} + \frac{D_o}{D_{ci}}\left(\frac{1}{\varepsilon_c} - 1\right)\right\}}$$

2) Useful heat for unit length

$$\frac{q_l}{L} = h_w(T_c - T_a)\pi D_{co} + \sigma \pi D_{co} \varepsilon_c (T_c^4 - T_{sky}^4)$$

3) Heat transfer coefficient between absorber tube and glass tube

$$\frac{k_{\text{eff}}}{k} = 0.317 (\text{Ra}^*)^{1/4}$$

$$(\text{Ra}^*)^{1/4} = \frac{\ln(D_{ci}/D_o)}{b^{3/4} \left(\frac{1}{D_o^{3/5}} + \frac{1}{D_{ci}^{3/5}}\right)^{5/4}} \text{Ra}^{1/4}$$

$$\frac{2\pi k_{\text{eff}}}{\ln(D_{ci}/D_o)} (T_{pm} - T_c) = h_{p-c} \pi D_o (T_{pm} - T_c)$$

$$h_{p-c} = \frac{2k_{\text{eff}}}{D_o \ln(D_{ci}/D_o)}$$

4) Heat transfer coefficient on the outer surface of the glass cover.

 $Nu = C_1 Re^n$ 

where  $C_1$  and *n* are constants having the following values:

For 40 < Re < 4000,  $C_1 = 0.615$ , n = 0.466For  $4000 < \text{Re} < 40\ 000$ ,  $C_1 = 0.174$ , n = 0.618For  $40\ 000 < \text{Re} < 400\ 000$ ,  $C_1 = 0.0239$ , n = 0.805

5) Take the air properties at  $104^{\circ}C$ 

At this temperature,

$$k = 0.0323 \text{ W/m-K}$$
  
 $v = 23.52 \times 10^{-6} \text{ m}^2/\text{s}$   
 $Pr = 0.688$ 

Equations for Compound parabolic collector.

1) Heat flux

$$S = \left[ I_b r_b + \frac{I_d}{C} \right] \tau \rho_e \alpha$$

2) Useful heat gain

$$q_{u} = F_{R}WL \left[ S - \frac{U_{l}}{C} (T_{fi} - T_{a}) \right]$$

$$F_{R} = \frac{\dot{m}C_{p}}{bU_{l}L} \left[ 1 - \exp\left\{ -\frac{F'bU_{l}L}{\dot{m}C_{p}} \right\} \right]$$

$$\frac{1}{F'} = U_{l} \left[ \frac{1}{U_{l}} + \frac{b}{N\pi D_{i}h_{f}} \right]$$

3) Tilt angle

$$r_b = \frac{\cos(L - \beta)\cos\delta\cos\omega + \sin(L - \beta)\sin\delta}{\cos L\cos\delta\cos\omega + \sin L\sin\delta}$$