Name:	ent No: UPES		
Enrolme	ent No:		
	UNIVERSITY OF PETROLEUM AND ENERGY STUDIES End Semester Examination, December 2018 : Design of Machine Elements Semester: V mme: B.Tech ADE		
Time: 0		: 100	
mstruc	SECTION A		
S. No.		Marks	CO
Q 1	Briefly summarize the following terminologies:i.Endurance limit.ii.Fatigue life.iii.Pitch circle of geariv.Diametral Pitchv.Module	10	CO2,C O4
Q2	Identify the various steps involved for the approximate estimation of endurance limit.	10	CO2
	SECTION B		
Q 3	A bracket is attached to the vertical column by means of six identical bolts as shown in fig. It is subjected to an eccentric force of 60 kN at a distance of 200 mm from the centre of the column. The maximum permissible shear for the bolt is 150 N/mm <sup>2</sup> . Determine the size of the bolts. $ \begin{array}{c}                                     $	15	CO3

	X = X $Y =$		
Q 4	Design a rigid type of flange coupling to connect two shafts of 36 mm diameter transmitting 15 kW at 720 rpm. The overload capacity is 1.25 times of the rated torque. Select the suitable material for the required components.	15	CO4
Q 5	A polished steel bar 30C8 ( $S_{ut}$ =1250 N/mm <sup>2</sup> ) is subjected to axial tensile force P. It has a groove 2 mm deep and having a radius of 3 mm. The notch sensitivity factor at the groove is .95. The outer diameter of the bar is 30 mm. The endurance limit in completely reversed bending is 600 MPa. Find the maximum force that the bar can carry for 10 <sup>5</sup> cycles with 90% reliability.	10	CO2
	SECTION-C		<u>.</u>
Q 6	A line shaft received power through a gear and pinion. The pinion is connected to an electric motor delivering 30 kW at 1200 rpm, of which 20 kW is supplied to a milling	20	CO4

machine through a horizontal pulley drive at P1 and the remainder of the power is supplied to a planer through pulley P2 by a vertical belt. The diameters of gear and pinion are 300 mm and 100 mm, respectively. The diameter of pulleys P1 and P2 are 750 mm and 900 mm respectively. The the layout of the shaft is shown in fig below and the ratio of belt tensions in both drives is 2.0, design the shaft on the basis of strength. (20) $\frac{150 \text{ mm}}{92} + \frac{300 \text{ mm}}{91} + \frac{150 \text{ mm}}{777} + \frac{92}{91} + \frac{100 \text{ mm}}{91} + \frac{150 \text{ mm}}{777} + \frac{92}{91} + \frac{100 \text{ mm}}{91} + \frac{100 \text{ mm}}{777} + \frac{92}{91} + \frac{100 \text{ mm}}{91} + \frac{100 \text{ mm}}{777} + \frac{92}{91} + \frac{100 \text{ mm}}{91} +$		
A shaft is supported on bearing A & B at both the end and 800 mm apart from each other. A 20 <sup>o</sup> straight tooth spur gear having 600 mm pitch diameter is located at 200 mm to the right of the left hand bearing A, and a 700 mm diameter pulley is mounted at 250 mm towards the left of bearing B. The gear is driven by a pinion with a downward tangential force while the pulley drives a horizontal belt having 180 <sup>o</sup> angle of wrap. The pulley also serves as a flywheel and weighs 3000 N. the maximum belt tension is 3000N and the tension ratio is 3. The shaft received 20 kW at 500 rpm and the load factor is 2.5. Design the shaft on ASME code basis also select the single row deep groove ball bearing if the expected life for 90% of the bearings is 8000h.		
Q 7 Design a spur gear to transmit the power of 10kW. The speed of driving motor and driven machines are 400 and 200 RPM. (20)	20	CO4

	factor	b	
Surface finish Ground	a 1.58	-0.085	Reliability R (%
Machined or cold-drawn	4.51	-0.265	50
Hot-rolled	57.7	-0.718	90
As forged	272	-0.995	
Vai	lues of size fi	actor	95
Diameter (d) (mm)	K <sub>h</sub>		99
<i>d</i> ≤7.5	1.00		99.9
$7.5 < d \le 50$	0.85		99.99
d > 50	0.75		99,999

Reliability factor

K<sub>c</sub> 1.000 0.897 0.868 0.814 0.753 0.702 0.659

Table: SKF bearing series 61

Designation	Basic load ratings (N)		Principal dimensions (mm)			Designation	Basic load ratings (N)		Principal dimensions (mm)		
•	C C <sub>0</sub>		B	D	d	2 congradient	Co	С	В	D	d
61808	3350	4160	7	52	40	61801	695	1430	5	21	12
16008	7800	13300	9	68	1000	6001	2240	5070	8	28	
6008	9300	16800	15	68		6201	3100	6890	10	32	
6208	16600	30700	18	80		6301	4650	9750	12	37	
6308	22400	41000	23	90							
6408	36500	63700	27	110		61802	815	1560	5	24	15
61809	3800	6050	7	58	45	6002	2500	5590	9	32	
16009	9300		10	58 75	45	6202	3550	7800	11	35	
		15600				6302	5400	11400	13	42	
6009 6209	12200 18600	21200 33200	16 19	75 85		61803	930	1680	5	26	17
6209		52700	25	100		6003	2800	6050	10	35	
6409	30000 45500	76100	29	120		6202	4500	9560	12	40	
0409	45500	/6100	29	120		6303	6550	13500	14	47	
61810	4250	6240	7	65	50	6403	11800	22900	17	62	
16010	10000	16300	10	80		61804	1500	2700	7	32	20
6010	13200	21600	16	80		16404	3400	7020	8	42	
6210	19600	35100	20	90		6004	4500	9360	12	42	
6310	36000	61800	27	110		6204	6200	12700	14	47	
6410	52000	87100	31	130		6304	7800	15900	15	52	
61811	5600	8320	9	72	55	6404	16600	30700	19	72	
16011	12200	19500	11	90		61805	1960	3120	7	37	25
6011	17000	28100	18	90		16005	4000	7610	8	47	
6211	25000	43600	21	100		6005	5600	11200	12	47	
6311	41500	71500	29	120		6205	6950	14000	15	52	
6411	63000	99500	33	140		6305	11400	22500	17	62	
			6.2.0		20152	6405	19600	35800	21	80	
61812	6100	8710	10	78	60	61806	2080	3120	7	42	30
16012	13200	19900	11	95		16006	5850	11200	9	42 55	30
6012	18300	29600	18	95					1.0		
6212	28000	47500	22	110		6006 6206	6800 10000	13300 19500	13 16	55 62	
6312	48000	81900	31	130			2.2.2.3		19	72	
6412	69500	108000	35	150		6306	14600	28100	23	90	
61813	8300	11700	10	85	65	6406	24000	43600			
16013	14600	21200	11	100		61807	3000	4030	7	47	35
6013	19600	30700	18	100		16007	6950	12400	9	62	
6213	34000	55900	23	120		6007	8500	15900	14	62	
6313	56000	92300	33	140		6207	13700	25500	17	72	
6413	78000	119000	37	160		6307	18000	33200	21	80	
(Conta			2011		<u>.</u>	6407	31000	55300	25	100	