


Name: Enrolment No:			
<p style="text-align: center;">UPES End Semester Examination, May 2025</p> <p> Course: Automotive Thermal Management Semester : 6th Program: B. Tech-Automotive Design Engineering Time : 03 hrs. Course Code: MECH-3027 Max. Marks: 100 </p> <p>Instructions: Assume any missing data and mention it clearly.</p>			
SECTION A (5Qx4M=20Marks)			
S. No.		Marks	CO
Q 1	Explain the Fourier's Law of heat conduction.	4	CO1
Q 2	What is Reynolds Number? State its importance.	4	CO2
Q 3	Distinguish between free and forced convective heat transfer by taking suitable example.	4	CO3
Q 4	Analyze the role of a black body in radiation heat transfer and evaluate how emissivity affects heat transfer in real-world systems. Compare and contrast black bodies and gray bodies, and provide examples of each in engineering applications, assessing their impact on system design.	4	CO4
Q 5	Evaluate the critical radius of insulation and explain why insulation might increase heat loss in some cases. Using a practical example of a pipe, analyze how the insulation thickness impacts heat transfer and justify when adding insulation reduces or increases heat loss.	4	CO5
SECTION B (4Qx10M= 40 Marks)			
Q 6	A small copper sphere of diameter 2 cm is initially at a temperature of 300°C and is suddenly immersed in a fluid at 50°C with a convective heat transfer coefficient of $h=120 \text{ W/m}^2 \text{ }^{\circ}\text{C}$ Assume properties of copper: <ul style="list-style-type: none"> $\rho=8950 \text{ kg/m}^3$ $c_p=385 \text{ J/kg K}$ $k=400 \text{ W/m }^{\circ}\text{C}$ Determine: <ol style="list-style-type: none"> Whether lumped system analysis is applicable. The temperature of the sphere after 60 seconds. 	10	CO2
Q 7	Deduce mathematical formulation for three-dimensional heat conduction equation with internal heat generation in cylindrical coordinates.	10	CO3
Q 8	Engine oil at 80°C flows over a flat surface at 40°C for cooling purpose, the flow velocity being 2 m/s. Determine the hydrodynamic and thermal	10	CO4

	boundary layer thickness at a distance of 0.4 m from the leading edge. Also determine the local and average values of friction and convection coefficients. Take the following properties kinetic viscosity = $83 \times 10^{-6} \text{ m}^2/\text{s}$, $\text{Pr} = 1050$. Thermal conductivity = 0.1407 W/m K .		
Q 9	<p>A straight aluminum fin of rectangular cross-section (thickness = 2 mm, width = 50 mm, length = 100 mm) is attached to a wall maintained at 150°C. The surrounding air is at 30°C, and the convective heat transfer coefficient is $h=60 \text{ W/m}^2 \text{ }^\circ\text{C}$. The thermal conductivity of aluminum is $k=200 \text{ W/m }^\circ\text{C}$. Assume steady-state, one-dimensional heat conduction with adiabatic tip.</p> <p>Determine:</p> <ol style="list-style-type: none"> 1. The fin efficiency. 2. Explain what the result implies about the fin's performance. <p style="text-align: center;">OR</p> <p>A cylindrical pin fin of length 5 cm and diameter 6 mm is used on a surface at 120°C, exposed to air at 25°C. The heat transfer coefficient is $h=45 \text{ W/m}^2 \text{ }^\circ\text{C}$, and the thermal conductivity of the fin material is $k=15 \text{ W/m }^\circ\text{C}$.</p> <p>Determine:</p> <ol style="list-style-type: none"> 1. The effectiveness of the fin. 2. Comment on whether adding this fin is beneficial. 	10	CO5
SECTION-C (2Qx20M=40 Marks)			
Q 10	<p>Consider a cylindrical furnace with radius and height of 1 m, as shown in figure. The top (surface 1) and the base (surface 2) of the furnace has emissivity $\epsilon_1=0.8$ and $\epsilon_2=0.4$, respectively, and are maintained at uniform temperatures $T_1=700 \text{ K}$ and $T_2=500 \text{ K}$. The side surface closely approximates a black body and is maintained at a temperature of $T_3=400 \text{ K}$. Determine the net rate of radiation heat transfer at each surface during steady operation and explain how these surfaces can be maintained at specified temperatures.</p> <div style="text-align: center;"> <p>The diagram shows a pink cylindrical furnace. The top surface is labeled with a circled 1, $T_1 = 700 \text{ K}$, and $\epsilon_1 = 0.8$. The bottom surface is labeled with a circled 2, $T_2 = 500 \text{ K}$, and $\epsilon_2 = 0.4$. The side surface is labeled with a circled 3, 'Black', and $T_3 = 400 \text{ K}$. The radius is labeled r_0 and the height is labeled H.</p> </div>	20	CO5

Q 11	<p>A counter-flow heat exchanger is used to heat cold water from 20°C to 60°C using hot oil that enters at 120°C and leaves at 80°C. The mass flow rates are:</p> <ul style="list-style-type: none"> Water: $\dot{m}_c=1$ kg/s, $C_{p_c}=4180$ J/kg °C Oil: $\dot{m}_h=2$ kg/s, $C_{p_h}=2100$ J/kg °C <p>Assume:</p> <ul style="list-style-type: none"> No heat loss to surroundings. Constant specific heats. <p>Determine</p> <ol style="list-style-type: none"> The effectiveness of the heat exchanger. The heat transfer rate. The required heat exchanger area if the overall heat transfer coefficient is $U=300$ W/m² °C <p style="text-align: center;">OR</p> <p>In a parallel-flow heat exchanger, hot gases at 300°C enter at a flow rate of 2 kg/s with $C_{p_h}=1000$ J/kg °C while cold water at 25°C enters at 1.5 kg/s with $C_{p_c}=4180$ J/kg °C</p> <p>After heat exchange:</p> <ul style="list-style-type: none"> Hot gas exits at 150°C Cold water exits at 100°C <p>Assume $U=200$ W/m² °C</p> <p>Determine</p> <ol style="list-style-type: none"> The rate of heat transfer. LMTD The required surface area of the heat exchanger 	20	CO5
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